

# Hierarchical Control of End-Point Impedance and Joint Impedance for Redundant Manipulators

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**Abstract**—This paper proposes an impedance control method for redundant manipulators, which can control not only the end-point impedance using one of the conventional impedance control methods, but the joint impedance which has no effects on the end-point impedance. First, a sufficient condition for the joint impedance controller is derived. Then, the optimal controller for a given desired joint impedance is designed using the least squares method. Finally, computer simulations and experiments using a planar direct-drive robot are performed in order to confirm the validity of the proposed method.

**Index Terms**—Impedance control, manipulators, redundancy, robot dynamics.

## I. INTRODUCTION

IMPEDANCE control [1] provides a unified approach for position and force control of a manipulator, where a controlled variable is a dynamic relation between motion and force. Using the motion equation of the manipulator and measurements of the manipulator's motion, this method can regulate the end-point impedance to the desired value which is designed depending on a given task. To date, many studies on impedance control have been conducted. Hogan [2] has proposed a method to implement impedance control without the calculation of an inverse of the Jacobian matrix. Tachi *et al.* [3] have developed an impedance control method that does not require the use of a force sensor. The robustness of impedance control for uncertainties such as modeling errors and disturbances has been discussed in [4], [5], and the stability of a manipulator performing a contact task has been analyzed in [6], [7].

In addition, a redundant manipulator which possesses more degrees of freedom than the ones required for performing a task has been considered to be the key to more dextrous and versatile robot motions. Many studies on arm redundancy have been performed in terms of the inverse kinematic problem [8]–[13], where arm redundancy is used in order to optimize criteria such as avoidance of singularity [10], avoidance of an obstacle [11]–[13], and various measures of dexterity [10], [12]. Also, redundancy in the force/torque transformation has been pointed out by Khatib [14], [15] and Kang and Freeman [16].

With regard to impedance control, a number of studies on the utilization of kinematic redundancy has been reported thus far [17]–[24]. Newman and Dohring [17] proposed the augmented impedance control method based on the extended Jacobian scheme developed by Baillieul [13]. In this scheme, a vector of new task variables with its dimension equal to the degrees of freedom of the kinematic redundancy is defined and augmented to the end-point position vector to make the Jacobian matrix square. By using the augmented Jacobian matrix, the impedance control law which satisfies a constraint expressed by the extended Jacobian matrix is derived. However, this method does not take inertial effects into account, so it reduces to the active stiffness control [25] rather than the impedance control. Later, Peng and Adachi [18] introduced a differentiable scalar objective function of joint angles, and derived the end-point impedance control law that optimizes the objective function. Using this control method, the kinematic redundancy can be utilized to control the arm configuration while controlling the end-point impedance. Also, Oh *et al.* [19] described the dynamics of the redundant manipulator in a task space including null space motion, and defined the desired inertia and viscosity for the null space dynamics as well as the hybrid impedance [26] for position and force controlled directions. The physical meaning of the impedance for the null space dynamics expressed in the task space, however, is not intuitively understood. Therefore, the planning of the desired null space impedance according to a given task may be a difficult problem. Also the stiffness is not included in the null space impedance parameters.

Our approach is significantly different from others. We have argued that the arm redundancy should be directly utilized in terms of the arm impedance itself [20], and proposed an impedance control method named *Multi-Point Impedance Control* (MPIC) [21]–[23]. The MPIC can control not only the end-point impedance using one of the conventional impedance control methods but also impedances of multiple points on the links of the manipulator utilizing arm redundancy. On the basis of the same idea, Liao and Donath [24] proposed the generalized impedance control for collision avoidance. Instead of the multi-point impedance, this paper proposes a method for controlling the joint impedance directly which has no effects on the end-point impedance.

Let us consider, as an example, a redundant manipulator performing a task that requires a compliant end-point motion as shown in Fig. 1(a), where the end-effector and a part of the arm are below the surface of the water. Various hydrodynamic forces caused by fluid such as the effects of added mass,

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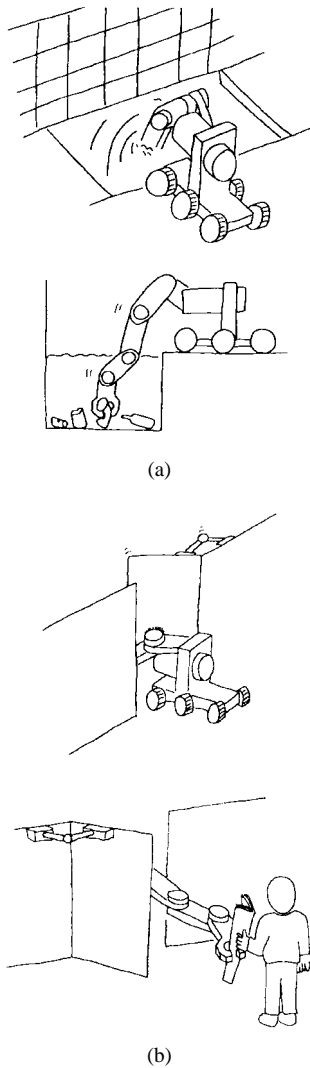


Fig. 1. Examples of tasks requiring impedance control of joints as well as the end-point.

drag and lift forces and buoyancy forces are exerted on the manipulator in underwater environments [27]. Although the previously proposed end-point impedance control methods can regulate the dynamic responses of the end-point motion for unknown external disturbances applied to the end-point, they are incompetent for disturbances applied to the links or the joints of the manipulators. In the presence of such disturbances, the joint configuration of the manipulator may change even if the end-point impedance is properly controlled, and it may result in serious accidents such as a collision with an obstacle or a singular joint configuration.

One possible solution to this problem is to set the impedance of specific joints under the water to be as large as possible in order to lock out the corresponding joints while controlling the end-point impedance to the given desired value. It could be possible to lock the underwater joints mechanically and use the conventional nonredundant impedance control for the rest of the joints. However, since both the kinematics and the dynamics of the manipulator are affected by locking the joints, the impedance control law should be modified depending on the number of locked joints and the arm posture which are not

always constant during the task. We need to develop a unified scheme that can control the joint impedance as well as the end-point impedance utilizing kinematic redundancy. Also the same problem may arise for a manipulator bracing against an object, e.g., a door, at a joint or a link as shown in Fig. 1(b). In this case, the impedance of the joints located between the base of the manipulator and the contact point with the object should be regulated to be as stiff as possible.

In this paper, a new impedance control method for redundant manipulators is developed, which can realize the closest joint impedance to the desired one in the least squares sense while still satisfying the desired end-point impedance. Under this impedance control, the desired joint impedance and the desired joint angle, that is, the equilibrium arm posture, can be specified, and the end-point impedance and the joint impedance are hierarchically controlled. Control of the joint impedance enables us to regulate the dynamic response of the joints for the unknown external forces beforehand.

This paper is organized as follows: in Section II, a joint impedance controller is incorporated into the end-point impedance controller in parallel, and a sufficient condition of the joint impedance controller so as not to affect the end-point impedance is derived. In Section III, the joint impedance controller corresponding to the desired joint impedance is derived using the least squares method. Computer simulations and experiments using a planar direct-drive robot are performed in Section IV in order to show the effectiveness of the proposed method.

## II. IMPEDANCE CONTROL UTILIZING KINEMATIC REDUNDANCY

In general, the motion equation of an  $m$ -joint manipulator can be expressed as

$$M(\theta)\ddot{\theta} + h(\theta, \dot{\theta}) = \tau + J^T(\theta)F_{\text{ext}} \quad (1)$$

where

$F_{\text{ext}} \in \mathfrak{R}^l$	external force exerted on the end-point;
$\theta \in \mathfrak{R}^m$	joint angle vector;
$M(\theta) \in \mathfrak{R}^{m \times m}$	nonsingular inertia matrix (hereafter denoted by $M$ );
$h(\theta, \dot{\theta}) \in \mathfrak{R}^m$	nonlinear term including the joint torque due to the centrifugal, Coriolis, gravitational, and frictional forces;
$\tau \in \mathfrak{R}^m$	joint torque vector;
$J(\theta) \in \mathfrak{R}^{l \times m}$	Jacobian matrix (hereafter denoted by $J$ );
$l$	dimension of the task space.

For a redundant manipulator,  $m$  is larger than  $l$ .

The desired impedance of the end-point is generally expressed as

$$M_e \ddot{X} + B_e d\dot{X} + K_e dX = F_{\text{ext}} \quad (2)$$

where  $M_e$ ,  $B_e$ ,  $K_e \in \mathfrak{R}^{l \times l}$  are the desired inertia, viscosity, and stiffness matrices of the end-point, respectively; and  $dX = X - X_d \in \mathfrak{R}^l$  is the deviation vector of the end-point position  $X$  from its equilibrium  $X_d$ .

In this paper, we adopt the impedance control law not using the inverse of the Jacobian matrix [2]:

$$\tau = \tau_{\text{effector}} + \tau_{\text{comp}} \quad (3)$$

$$\tau_{\text{effector}} = -J^T [\Lambda \{M_e^{-1} (K_e dX + B_e d\dot{X}) + \dot{J}\dot{\theta}\} + \{I - \Lambda M_e^{-1}\} F_{\text{ext}}] \quad (4)$$

$$\tau_{\text{comp}} = (J_{M-1}^+ J)^T \hat{h}(\theta, \dot{\theta}) \quad (5)$$

where  $\Lambda = (J\hat{M}^{-1}J^T)^{-1} \in \mathbb{R}^{l \times l}$  is the operational space kinetic energy matrix [14], [15];  $J_{M-1}^+ = \hat{M}^{-1}J^T\Lambda \in \mathbb{R}^{m \times l}$  the generalized inverse of  $J$  weighted by  $\hat{M}^{-1}$ ;  $\tau_{\text{effector}} \in \mathbb{R}^m$  the joint torque vector needed to produce the desired end-point impedance;  $I$  the  $l \times l$  unit matrix; and  $\tau_{\text{comp}} \in \mathbb{R}^m$  the joint torque vector for nonlinear compensation. Also,  $\hat{M}(\theta)$  and  $\hat{h}(\theta, \dot{\theta})$  are the estimated values of  $M(\theta)$  and  $h(\theta, \dot{\theta})$ . It is assumed in this paper that the joint configuration  $\theta$  is not in a singular posture, and  $\hat{M}(\theta) = M(\theta)$  and  $\hat{h}(\theta, \dot{\theta}) = h(\theta, \dot{\theta})$  are held.

Although the control law (3)–(5) and other impedance control methods such as [1], [3] can be applied to redundant manipulators, arm redundancy cannot be effectively utilized. In this paper, an additional controller is incorporated into the end-point impedance control law in order to exploit the arm redundancy

$$\tau = \tau_{\text{add}} + \tau_{\text{effector}} + \tau_{\text{comp}} \quad (6)$$

where  $\tau_{\text{add}} \in \mathbb{R}^m$  is the joint control torque vector for an additional purpose. It can be easily shown that if the additional joint control torque  $\tau_{\text{add}}$  satisfies the equation

$$(J_{M-1}^+)^T \tau_{\text{add}} = 0 \quad (7)$$

then  $\tau_{\text{add}}$  has no dynamical effects on the end-point motion of the manipulator, so that the end-point impedance remains equal to the desired one given by (2). This force redundancy was found by Khatib [14], [15], after which Kang and Freeman [16] used the force redundancy to assure stability in a joint torque optimization for the redundant manipulator by introducing the null-space damping technique. In the following sections, we present a method to utilize the force redundancy in terms of the joint impedance control.

### III. JOINT IMPEDANCE REGULATION

#### A. Optimal Additional Controller

Equation (7) gives the sufficient condition for the additional joint torque  $\tau_{\text{add}}$  to have no effect on the end-point's motion. In this subsection, we assume that the desired value of the additional torque  $\tau_{\text{add}}^*$  is given, which does not always satisfy (7). We then need to derive  $\tau_{\text{add}}$  that satisfies (7) and is the closest to the desired joint torque  $\tau_{\text{add}}^*$ .

The general solution of (7) is given by

$$\tau_{\text{add}} = \{I_m - \Omega (J_{M-1}^+)^T\} z_1 \quad (8)$$

where  $I_m$  is the  $m \times m$  unit matrix;  $z_1 \in \mathbb{R}^m$  an arbitrary constant vector; and  $\Omega = J_{M-1}^+ \{(J_{M-1}^+)^T J_{M-1}^+\}^{-1} \in \mathbb{R}^{m \times l}$  the right-side generalized inverse of  $(J_{M-1}^+)^T$ . The additional

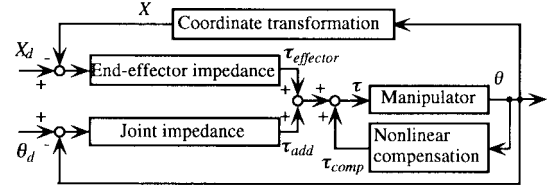


Fig. 2. Hierarchical impedance control for redundant manipulators.

joint torque  $\tau_{\text{add}}$  in (8) always satisfies the sufficient condition (7). The vector  $z_1$  is then chosen in such a way that the following cost function  $G_1(z_1)$  is minimized:

$$G_1(z_1) = \|W \{\tau_{\text{add}}^* - \tau_{\text{add}}(z_1)\}\| \quad (9)$$

where  $W \in \mathbb{R}^{m \times m}$  is a diagonal positive definite matrix, each diagonal element of which is a weight to the corresponding element of the joint torque error vector  $\tau_{\text{add}}^* - \tau_{\text{add}}$ , and  $\|a\|$  stands for the Euclidian norm of the vector  $a$

$$\|a\| = (a^T a)^{1/2} = \left( \sum_{i=1}^m a_i^2 \right)^{1/2} \quad (10)$$

where  $a_i$  is the  $i$ th element of vector  $a \in \mathbb{R}^m$ . Using the least squares method, we can find the optimal solution given as

$$\tau_{\text{add}} = \Gamma \tau_{\text{add}}^* \quad (11)$$

$$\Gamma = I_m - \{\Omega_{W^{-2}} (J_{M-1}^+)^T\} \quad (12)$$

where  $\Omega_{W^{-2}} = W^{-2} J_{M-1}^+ \{J_{M-1}^+ W^{-2} (J_{M-1}^+)^T\}^{-1} \in \mathbb{R}^{m \times l}$  is the right-side generalized inverse of  $(J_{M-1}^+)^T$  weighted by  $W^{-2} \equiv W^{-1} W^{-1}$ . As a result, using (4)–(6), (11), and (12), we can control not only the end-point impedance but the joint torque which is the closest to the given desired one without any effects on the end-point's motion.

#### B. Joint Impedance Control

It is then assumed that the desired joint impedance is given according to the task. The desired joint torque for the joint impedance regulation can be defined as

$$\tau_{\text{add}}^* = -M_j^* \ddot{\theta} - B_j^* \dot{\theta} - K_j^* d\theta \quad (13)$$

where  $M_j^*$ ,  $B_j^*$ ,  $K_j^* \in \mathbb{R}^{m \times m}$  are the desired joint inertia, viscosity, and stiffness matrices, respectively, and  $d\theta = \theta - \theta_d \in \mathbb{R}^m$  is the deviation vector between the joint angle  $\theta$  and its equilibrium one  $\theta_d$ . Using (11)–(13), we can control the joint impedance as a subtask which has no effects on the end-point's motion (see Fig. 2).

The joint impedance realized by this control scheme, however, may differ from the desired one, since the proposed impedance control does not always achieve the desired joint torque. The property of the realized joint impedance is analyzed in the rest of this section.

First, the realized joint impedance related with the additional joint torque is expressed as

$$\tau_{\text{add}} = -M_j \ddot{\theta} - B_j \dot{\theta} - K_j d\theta \quad (14)$$

where  $M_j$ ,  $B_j$ ,  $K_j \in \mathbb{R}^{m \times m}$  are the realized joint inertia, viscosity and stiffness matrices, respectively. It is easily seen

that the sufficient condition (7) is always satisfied if the following conditions are given:

$$(J_{M-1}^+)^T M_j = 0, \quad (15)$$

$$(J_{M-1}^+)^T B_j = 0 \quad (16)$$

$$(J_{M-1}^+)^T K_j = 0. \quad (17)$$

For the joint inertia matrix  $M_j$ , the general solution of the matrix (15) is given as

$$M_j = \{I_m - \Omega(J_{M-1}^+)^T\} Z_2 \quad (18)$$

where  $Z_2 \in \mathfrak{R}^{m \times m}$  is an arbitrary constant matrix. The desired joint inertia matrix  $M_j^*$  is already given, so the problem is how to find an arbitrary constant matrix  $Z_2$  in (18) which results in  $M_j$  close to  $M_j^*$ . To solve this problem, the following cost function  $G_2(Z_2)$  is minimized with respect to  $Z_2$ :

$$G_2(Z_2) = \|W\{M_j^* - M_j(Z_2)\}\| \quad (19)$$

where  $W \in \mathfrak{R}^{m \times m}$  is a diagonal positive definite matrix, each diagonal element of which acts as a weight for the corresponding row of the error matrix between  $M_j^*$  and  $M_j$ . The  $\|A\|$  stands for the matrix norm defined by

$$\|A\| = (\text{tr}[A^T A])^{1/2} = \left( \sum_{i=1}^m \sum_{j=1}^m a_{ij}^2 \right)^{1/2} \quad (20)$$

where  $A = [a_{ij}] \in \mathfrak{R}^{m \times m}$  is an arbitrary matrix; and  $\text{tr}[\cdot]$  denotes the trace of the matrix. Using the least squares method, we can find the optimal solution given as (see Appendix A)

$$M_j = \Gamma M_j^* \quad (21)$$

where  $\Gamma$  is given in (12). In the same manner, we can also derive the optimal solutions for the joint viscosity matrix  $B_j$  and the joint stiffness matrix  $K_j$ :

$$B_j = \Gamma B_j^*, \quad (22)$$

$$K_j = \Gamma K_j^*. \quad (23)$$

Obviously, the additional joint control torque (14) with  $M_j$ ,  $B_j$ ,  $K_j$  defined by (21)–(23) is exactly the same as the control law (11)–(13). This means that the proposed method always realizes the optimal joint impedance in a sense that the realized joint impedance is the closest to the desired joint impedance while achieving the desired end-point impedance.

### C. Positive Semidefinite Joint Impedance

For a redundant manipulator, even if the stability of the end-point impedance is guaranteed, it does not mean that the joint motion is always stable. In this subsection, the joint impedance controller that assures the stability of the joint motion is derived on the basis of the positive semidefiniteness of the joint impedance matrices  $M_j$ ,  $B_j$ , and  $K_j$ .

In general, the positive semidefiniteness of  $M_j \in \mathfrak{R}^{m \times m}$  can be assured if it is expressed in the form

$$M_j = D_j D_j^T \quad (24)$$

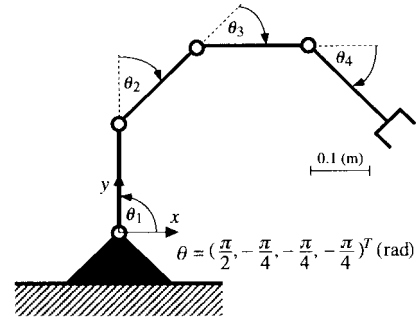


Fig. 3. Four-joint planar manipulator used in computer simulations.

where  $D_j$  is an  $m \times m$  matrix. Substituting (24) into (15), we obtain

$$(J_{M-1}^+)^T D_j D_j^T = 0. \quad (25)$$

Since  $D_j$  is not a zero matrix, the following equation must hold:

$$(J_{M-1}^+)^T D_j = 0. \quad (26)$$

The desired joint inertia matrix  $M_j^*$  is naturally assumed to be given as a positive semidefinite matrix, so it is also expressed as  $M_j^* = (D_j^*)(D_j^*)^T$ . Using the weighting matrix  $W$  and replacing  $M_j$  with  $D_j$ , we can derive the optimal solution  $D_j$  of the matrix equation (26) in the same way as described in the previous subsection:

$$D_j = \Gamma D_j^*. \quad (27)$$

Then, substituting (27) into (24) and using  $M_j^* = (D_j^*)(D_j^*)^T$ , we can obtain the following optimal joint inertia matrix:

$$M_j = \Gamma M_j^* \Gamma^T \quad (28)$$

which is the closest to the desired inertia  $M_j^*$  in the class of positive semidefinite matrices satisfying (15).

Also the joint viscosity matrix  $B_j$  and the joint stiffness matrix  $K_j$  can be derived as

$$B_j = \Gamma B_j^* \Gamma^T, \quad (29)$$

$$K_j = \Gamma K_j^* \Gamma^T. \quad (30)$$

Consequently, the additional joint controller (14) with the inertia, viscosity and stiffness matrices given in (28)–(30) guarantees the stability of the joint motion of the manipulator.

## IV. COMPUTER SIMULATIONS AND EXPERIMENTS

### A. Numerical Examples

The realized joint stiffness matrix  $K_j$  was computed using the proposed method for a four-joint planar manipulator of Fig. 3, where the link parameters of the manipulator are shown in Table I.

Table II shows the computed joint stiffness matrices and the index  $E(K_j)$  which is defined in order to measure how close the realized joint stiffness  $K_j$  approaches the desired one  $K_j^*$

$$E(K_j) = (\text{tr}[(K_j^* - K_j)^T (K_j^* - K_j)])^{1/2}. \quad (31)$$

From Table II, it can be seen that the realized joint stiffness matrices reflect the corresponding desired ones. Also for

TABLE I  
 LINK PARAMETERS OF THE FOUR-JOINT PLANAR MANIPULATOR OF FIG. 2

	Link $i$ ( $i = 1,2,3,4$ )
Length (m)	0.20
Mass (kg)	1.57
Center of mass (m)	0.10
Moment of inertia ( $\text{kgm}^2$ )	0.80

 TABLE II  
 JOINT STIFFNESS MATRICES REALIZED BY THE PROPOSED METHOD

Types	Desired joint stiffness matrix, $K_j$ (Nm/rad)	Weighting matrix, $W$	Realized joint stiffness matrix, $K_j$ (Nm/rad)
(a) Hierarchical impedance control, Eq. (23)	$\begin{bmatrix} 100.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 10.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 10.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 100.0 \end{bmatrix}$	$\begin{bmatrix} 1.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 1.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 1.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 1.0 \end{bmatrix}$	$\begin{bmatrix} 58.0 & 0.3 & 2.7 & 41.1 \\ 3.2 & 2.7 & -3.6 & 25.1 \\ 27.1 & -3.6 & 6.6 & -13.2 \\ 41.1 & 2.5 & -1.3 & 48.7 \end{bmatrix}$ $E(K_j) = 97.1$ (Nm/rad)
	$\begin{bmatrix} 100.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 10.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 10.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 100.0 \end{bmatrix}$	$\begin{bmatrix} 1.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 1.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 1.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 1.0 \end{bmatrix}$	$\begin{bmatrix} 3.7 & -11.7 & 0.4 & 142.5 \\ -1.2 & 9.4 & 0.2 & -3.4 \\ 3.6 & -19.7 & 0.5 & 72.8 \\ 1.4 & 0.3 & 0.1 & 97.2 \end{bmatrix}$ $E(K_j) = 188.6$ (Nm/rad)
(b) Hierarchical impedance control with positive semi-definite joint impedance, Eq. (30)	$\begin{bmatrix} 100.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 10.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 10.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 100.0 \end{bmatrix}$	$\begin{bmatrix} 1.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 1.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 1.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 1.0 \end{bmatrix}$	$\begin{bmatrix} 51.3 & 11.3 & 11.9 & 43.6 \\ 11.3 & 8.4 & -5.8 & 14.7 \\ 11.9 & -5.8 & 14.8 & 2.9 \\ 43.6 & 14.8 & 2.9 & 41.5 \end{bmatrix}$ $E(K_j) = 103.4$ (Nm/rad)
	$\begin{bmatrix} 100.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 10.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 10.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 100.0 \end{bmatrix}$	$\begin{bmatrix} 1.0 & 0.0 & 0.0 & 0.0 \\ 0.0 & 1.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 1.0 & 0.0 \\ 0.0 & 0.0 & 0.0 & 1.0 \end{bmatrix}$	$\begin{bmatrix} 216.9 & -6.2 & 127.1 & 138.3 \\ -6.2 & 9.0 & -16.2 & 3.6 \\ 127.1 & -16.2 & 92.2 & 70.2 \\ 138.3 & 3.6 & 70.2 & 94.6 \end{bmatrix}$ $E(K_j) = 318.6$ (Nm/rad)

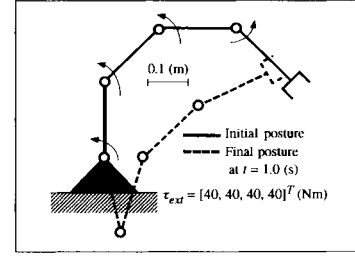
the large diagonal elements of the weighting matrix  $W$ , the elements in the corresponding row of  $K_j$  approach closer to the desired values than the ones in other rows. It should be noted that  $K_j$  in Table II(a) is not symmetric. On the other hand, as seen in Table II(b),  $K_j$  is always symmetric and positive semidefinite, although the discrepancy between  $K_j^*$  and  $K_j$  increases as indicated by the value of the index  $E(K_j)$ .

### B. Computer Simulations

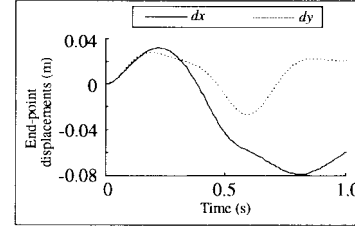
In the next step, computer simulations were performed using the four-joint planar manipulator shown in Fig. 3, where the desired end-point impedance matrices were  $M_e = \text{diag.}[1, 1]$  (kg),  $B_e = \text{diag.}[20, 10]$  (Ns/m),  $K_e = \text{diag.}[100, 400]$  (N/m). The damping ratios of the desired dynamic response for the end-point in the direction of  $x$  and  $y$  axes were 1.0 and 0.25, respectively, and the settling times of the end-point in the direction of  $x$  and  $y$  axes were 0.4 (s) and 0.8 (s), respectively. Also, in the proposed method, the desired joint impedance matrices were set as  $M_j^* = \text{diag.}[0.1, 0.1, 0.1, 0.1]$  ( $\text{kgm}^2$ ),  $B_j^* = \text{diag.}[80, 8, 80, 8]$  (Nms/rad) and  $K_j^* = \text{diag.}[4000, 40, 4000, 40]$  (Nm/rad); and the weighting matrix was  $W = \text{diag.}[50, 1, 50, 1]$ . Moreover the desired end-point position  $X_d$  and the equilibrium joint angle  $\theta_d$  were chosen as  $X_d(t) = X(0)$  and  $\theta_d(t) = \theta(0)$ , respectively. The computations of the manipulator dynamics were performed using Appel's method [28].

Figs. 4 and 5 show the simulation results of the manipulator under the conventional impedance control method (3)–(5) and the proposed method (4)–(6), (14), (28)–(30), respectively, where the external torque disturbance,  $\tau_{\text{ext}} = [40, 40, 40, 40]^T$  (Nm), is exerted to the joints of the manipulator.

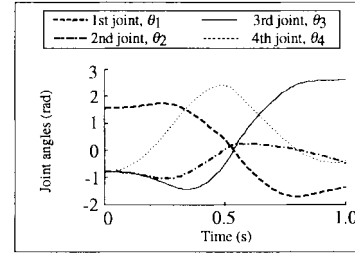
In Fig. 4, all joints rotated and oscillated according to the torque disturbance, since no viscous friction was assumed for the rotation of all joints. It can be seen that the conventional



(a)



(b)



(c)

Fig. 4. Motion profile of the four-joint manipulator for the external torque exerted to the joints of the manipulator under the conventional impedance control: (a) stick pictures, (b) end-point displacements, and (c) joint angles.

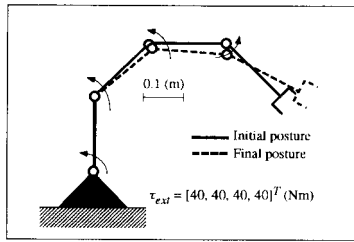
end-point impedance controller used in this paper cannot resist the external joint torque disturbance. On the other hand, the first and third joints did not rotate in Fig. 5, since the desired impedance of the corresponding joints were set to be large. The proposed method is effective in resisting the unknown external joint torque disturbance.

Next, the proposed method was applied to a crank rotation task by a four-joint planar manipulator of Fig. 6. The link parameters of the manipulator are shown in Table III. Two kinds of coordinate systems were chosen:

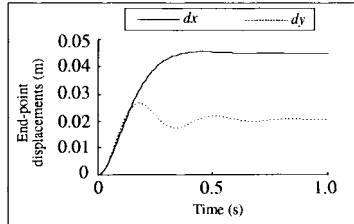
- 1) world coordinate system  $X(x, y)$ ;
- 2) polar coordinate system  $\Phi(\phi, r)$  with its origin at the center of the crank, where  $\phi$  is the rotational angle of the crank and  $r$  is the distance from the center of the crank to the end-point, i.e., the radius of the crank.

In Fig. 6,  $r = 0.15$  m was used. Also, the viscous friction of each joint was set to 10.0 Nms/rad in order to avoid unstable results of the conventional impedance control.

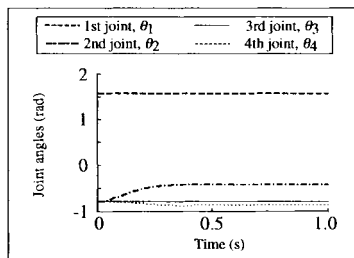
The desired end-point impedance was expressed in the polar coordinate system, where the inertia, viscosity and stiffness matrices were chosen as  $M_e = \text{diag.}[2.25 \times 10^{-3}$  ( $\text{kgm}^2$ ), 0.1 (kg)];  $B_e = \text{diag.}[0.45$  (Nms/rad), 2 (Ns/m)]; and  $K_e = \text{diag.}[22.5$  (Nm/rad), 10 (N/m)]. Also, the desired end-point



(a)



(b)



(c)

Fig. 5. Motion profile of the four-joint manipulator for the external torque exerted to the joints of the manipulator under the proposed method: (a) stick pictures, (b) end-point displacements, and (c) joint angles.

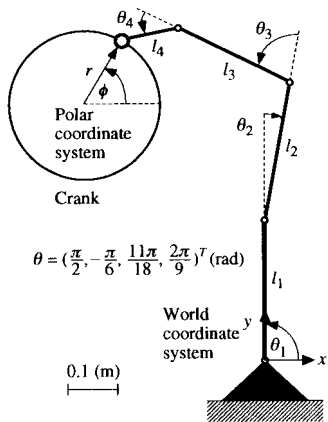


Fig. 6. The four-joint planar manipulator performing a crank rotation task.

trajectory, that is, the equilibrium trajectory, was defined as

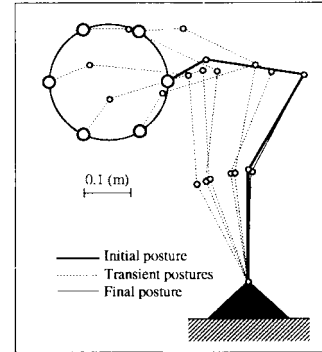
$$\begin{bmatrix} \phi_d(t) \\ r_d(t) \end{bmatrix} = \begin{bmatrix} 20\pi t^3/t_f^3 - 30\pi t^4/t_f^4 + 12\pi t^5/t_f^5 \\ r \end{bmatrix} \quad (32)$$

where the time duration  $t_f$  was set to 2.0 s.

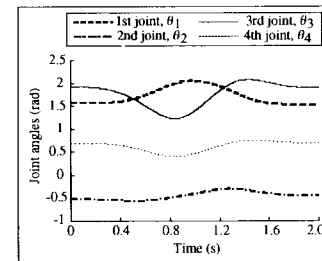
Figs. 7 and 8 show simulation results under the conventional impedance control method and the proposed method, respectively. The desired joint impedance matrices in (13) were set as  $M_j^* = \text{diag.} [0.001, 0.0001, 0.0001, 0.001]$  (kgm<sup>2</sup>);  $B_j^*$

TABLE III  
LINK PARAMETERS OF THE FOUR-JOINT PLANAR MANIPULATOR OF FIG. 5

	Link 1, 2	Link 3	Link 4
Length (m)	0.28	0.25	0.11
Mass (kg)	3.392	1.92	1.152
Center of mass (m)	0.128	0.1025	0.055
Moment of inertia (kgm <sup>2</sup> )	0.29312	0.011017	0.00123



(a)



(b)

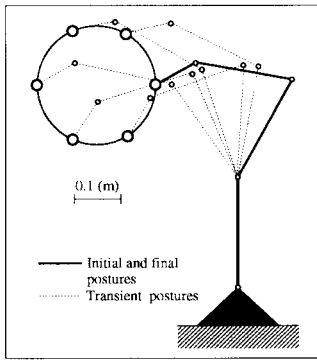
Fig. 7. Motion profile of the four-joint manipulator during the crank rotation task under the conventional impedance control: (a) stick pictures and (b) joint angles.

= diag. [2, 0.2, 0.2, 2] (Nms/rad); and  $K_j^* = \text{diag.} [1000, 100, 100, 1000]$  (Nm/rad) with the equilibrium joint trajectory  $\theta_d(t) = \theta(0)$ . The weighting matrix was set as  $W = \text{diag.} [50, 1, 1, 50]$  in order to assign high priority to the impedance of the first and fourth joints.

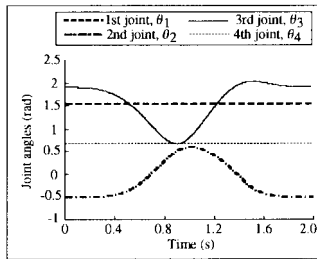
Although the manipulator can rotate the crank properly in both the cases, the effectiveness of the proposed method appears clearly in the joint motion. The first and fourth joints did not move during the crank rotation under the proposed method, since their impedances were set to be larger than the ones of other joints. As a result, the initial and final joint configurations were exactly the same, and the end-point motion was realized by using only the second and third joints.

### C. Experiments

Finally, experiments using a three-joint direct-drive manipulator of Fig. 9 were performed, where the link parameters are shown in Table IV. The joint angles were measured by means of optical encoders (resolution:  $1.745 \times 10^{-5}$  rad) and the control law was implemented by using four CPU's (Transputer, T800, 25 MHz).



(a)



(b)

Fig. 8. Motion profile of the four-joint manipulator during the crank rotation task under the proposed method. The impedances of the first and fourth joints are set considerably larger than the ones of other joints: (a) stick pictures and (b) joint angles.

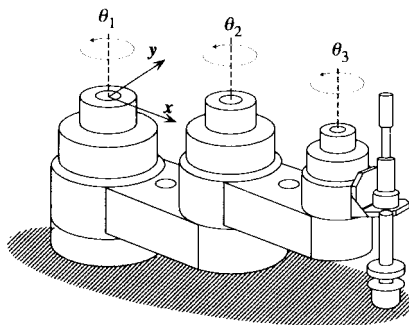
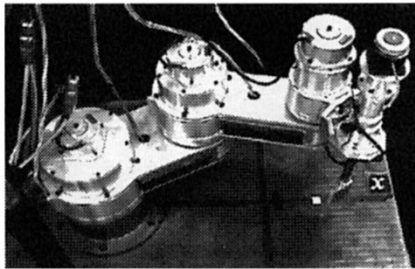
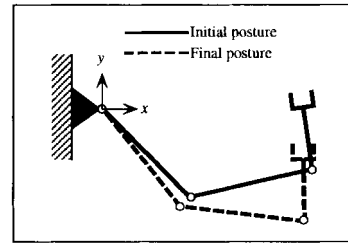


Fig. 9. Direct-drive robot used in the experiments.

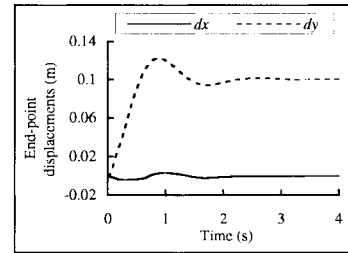
Figs. 10–12 show the experimental results, where the desired end-point impedance matrices were  $M_e = \text{diag.}[25, 25]$  (kg),  $B_e = \text{diag.}[80, 80]$  (Ns/m), and  $K_e = \text{diag.}[400, 400]$  (N/m). The damping ratio of the desired dynamic response of the end-point in each direction was 0.4. In Figs. 10 and 11, only the end-point impedance was controlled under the

TABLE IV  
LINK PARAMETERS OF THE DIRECT-DRIVE ROBOT

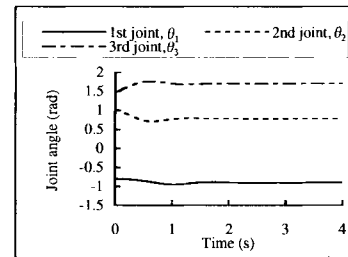
	Link 1	Link 2	Link 3
Length (m)	0.25	0.25	0.125
Mass (kg)	20.8	13.2	8.84
Center of mass (m)	0.064	0.065	0.031
Moment of inertia (kgm <sup>2</sup> )	0.334	0.196	0.0851
Joint friction (Nms/rad)	2.69	1.88	0.0634



(a)



(b)



(c)

Fig. 10. Step response of the direct-drive robot under the conventional impedance control: (a) stick pictures, (b) end-point displacements, and (c) joint angles.

conventional impedance control (3)–(5), while the proposed method (4)–(6), (14), (28)–(30) was used in Fig. 12. For the proposed method, the desired joint impedance matrices were set as  $M_j^* = \text{diag.}[0.1, 0, 0]$  (kgm<sup>2</sup>),  $B_j^* = \text{diag.}[20, 0, 0]$  (Nms/rad), and  $K_j^* = \text{diag.}[1000, 0, 0]$  (Nm/rad). The unit matrix was used as the weight matrix  $W$  for simplicity. Also the desired end-point position  $X_d(t)$  was 0.1 m in the direction of  $x$  axis from its initial position, while the equilibrium joint angle was fixed at the initial angle  $\theta_d(t) = \theta(0)$ . The sampling interval of the controller was 1.4 ms for both the conventional impedance control and the proposed method.

Both methods realized almost the same end-point trajectories as the one prespecified by the desired end-point

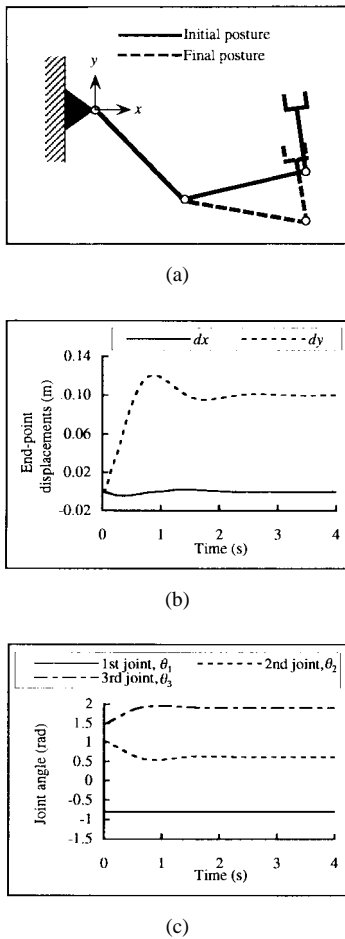


Fig. 11. Step response of the direct-drive robot under the proposed method, where the stiffness of the first joint is regulated considerably larger than the ones of other joints: (a) stick pictures, (b) end-point displacements, and (c) joint angles.

impedance. However, one of the joints was almost fixed in Figs. 11 and 12, which cannot be realized using any conventional impedance control method.

## V. CONCLUSIONS

In this paper, we have proposed a new impedance control method that can effectively utilize arm redundancy in terms of the arm impedance by incorporating an additional controller to the end-point impedance controller. The distinctive feature of the proposed method is the ability to control not only the end-point impedance but also the joint impedance in such a way that it has no effects on the end-point motion of the manipulator. When the desired joint impedance is chosen adequately according to the task, the proposed method can realize the closest joint impedance in the least squares sense while still satisfying the desired end-point impedance.

Experimental verification of the proposed method has been done by using the direct-drive robot with sufficient accuracy. In order to reduce the control error, future research will be directed to develop a robust impedance control that can realize the desired arm impedance when the model of the manipulator is not precisely given.

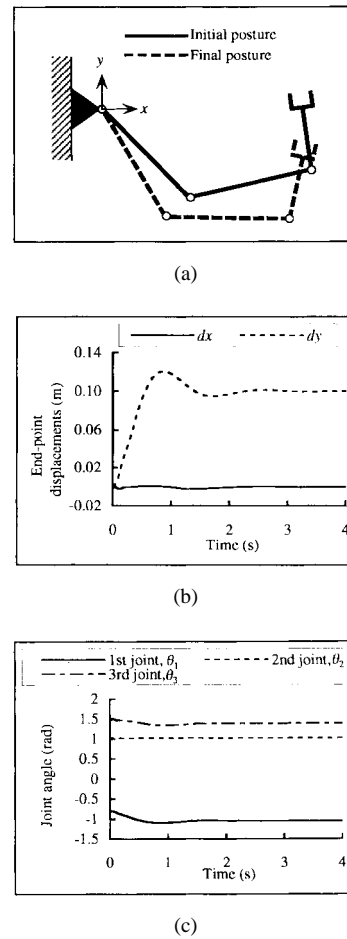


Fig. 12. Step response of the direct-drive robot under the proposed method, where the stiffness of the second joint is set considerably larger than the ones of other joints: (a) stick pictures, (b) end-point displacements, and (c) joint angles.

## APPENDIX

Let us consider the case of  $W = I_m$  as the first step. In this case, the objective function (19) reduces to

$$G_1(Z_2) = \|M_j^* - M_j(Z_2)\|. \quad (33)$$

Substituting (18) into (33), we find

$$G_2(Z_2) = (\text{tr}[[M_j^* - \{I_m - \Omega(J_{M-1}^+)^T\}Z_2]^T \cdot [M_j^* - \{I_m - \Omega(J_{M-1}^+)^T\}Z_2]])^{0.5}. \quad (34)$$

The problem is how to find the matrix  $Z_2$  that minimizes the objective function  $G_2(Z_2)$ . The necessary condition with regard to the optimal solution of the above problem is given by

$$\frac{\partial G_2(Z_2)}{\partial Z_2} = 0. \quad (35)$$

Substituting (34) into (35) and expanding it, we have

$$\{I_m - \Omega(J_{M-1}^+)^T\}Z_2 = \{I_m - \Omega(J_{M-1}^+)^T\}M_j^*. \quad (36)$$



Note that the partial differential formulas for the trace of matrices [29]

$$\frac{\partial \text{tr}[PZ_2Q]}{\partial Z_2} = P^T Q^T, \quad (37)$$

$$\frac{\partial \text{tr}[PZ_2^T QZ_2 R]}{\partial Z_2} = QZ_2 R P + Q^T Z_2 P^T R^T \quad (38)$$

and the property of  $A^+$ , that is,  $(A^+A)^T(A^+A) = (A^+A)^T = (A^+A)$ , is used in the derivation of (36). Here,  $P$ ,  $Q$ , and  $R$  are matrices with appropriate dimensions. Then, substituting (36) into (18), we obtain

$$M_j = \{I_m - \Omega(J_{M-1}^+)^T\} M_j^*. \quad (39)$$

The above equation gives the least squares solution of the matrix equation (15) with the objective function (33).

Next, we will derive the optimal solution for the general case, where the weighting matrix  $W$  is not the unit matrix. In this case, we rewrite the matrix equation (15) as follows:

$$(J_{M-1}^+)^T W^{-1} W M_j = (W^{-1} J_{M-1}^+)^T W M_j = 0. \quad (40)$$

Comparing (40) with (15), we can see that (40) has the same form as (15), in which  $W^{-1} J_{M-1}^+$  and  $W M_j$  are corresponding to  $J_{M-1}^+$  and  $M_j$  in (15), respectively. Therefore, the general solution of (40) for  $W M_j$  can be easily derived as

$$W M_j = [I_m - \{(W^{-1} J_{M-1}^+)^T\}^+ (W^{-1} J_{M-1}^+)^T] Z_3 \quad (41)$$

where  $\{(W^{-1} J_{M-1}^+)^T\}^+ = W^{-1} J_{M-1}^+ \{(J_{M-1}^+)^T W^{-2} J_{M-1}^+\}^{-1} \in \mathfrak{R}^{m \times l}$  is the right-side generalized inverse of  $(W^{-1} J_{M-1}^+)^T$  and  $Z_3 \in \mathfrak{R}^{m \times m}$  is an arbitrary constant matrix.

Substituting (41) into (19) and finding the matrix  $Z_3$  that minimizes the cost function, we have

$$\begin{aligned} & [I_m - \{(W^{-1} J_{M-1}^+)^T\}^+ (W^{-1} J_{M-1}^+)^T] Z_3 \\ &= [I_m - \{(W^{-1} J_{M-1}^+)^T\}^+ (W^{-1} J_{M-1}^+)^T] W M_j^*. \end{aligned} \quad (42)$$

Finally, substituting (42) into (41) and expanding it, we can get

$$\begin{aligned} M_j &= W^{-1} [I_m - \{(W^{-1} J_{M-1}^+)^T\}^+ (W^{-1} J_{M-1}^+)^T] W M_j^* \\ &= \{I_m - \Omega_{W^{-2}}(J_{M-1}^+)^T\} M_j^* \\ &= \Gamma M_j^*, \end{aligned} \quad (43)$$

where  $\Omega_{W^{-2}} = W^{-2} J_{M-1}^+ \{J_{M-1}^+ W^{-2} (J_{M-1}^+)^T\}^{-1} \in \mathfrak{R}^{m \times l}$  is the right-side generalized inverse of  $J_{M-1}^+$  weighted by  $W^{-2} \equiv W^{-1} W^{-1}$ . Consequently, the objective function (19) is minimized by the optimal joint inertia matrix  $M_j$  of (21) as derived above.

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